

EM311M Notes

1 Kinematics of a particle

Definitions

position vector

$$\mathbf{r} = \mathbf{r}(t)$$

velocity vector

$$\mathbf{v} = \dot{\mathbf{r}} = \frac{d\mathbf{r}}{dt}$$

speed

$$|\mathbf{v}|$$

acceleration vector

$$\mathbf{a} = \dot{\mathbf{v}} = \ddot{\mathbf{r}}$$

tangential and normal acceleration vectors

$$\mathbf{a}_t = \left(\mathbf{a} \cdot \frac{\mathbf{v}}{|\mathbf{v}|} \right) \frac{\mathbf{v}}{|\mathbf{v}|}$$
$$\mathbf{a}_n = \mathbf{a} - \mathbf{a}_t$$

Cartesian coordinates

$$\mathbf{r} = x\mathbf{i} + y\mathbf{j} + z\mathbf{k}$$

$$\mathbf{v} = \dot{\mathbf{r}} = (x\mathbf{i} + y\mathbf{j} + z\mathbf{k})' = \underbrace{\dot{x}}_{v_x} \mathbf{i} + \underbrace{\dot{y}}_{v_y} \mathbf{j} + \underbrace{\dot{z}}_{v_z} \mathbf{k}$$

So $a_x = \dot{v}_x = \ddot{x}, \dots$

Rectilinear motion

$$\mathbf{r} = x\mathbf{i}$$

$$s := x$$

$$v := v_x = \dot{x} = \dot{s}$$

$$a := a_x = \dot{v} = \ddot{s}$$

$$a = \frac{dv}{dt}, v = \frac{ds}{dt} \Rightarrow a ds = v dv$$

Cylindrical coordinates

$$\begin{cases} x = r \cos \theta \\ y = r \sin \theta \\ z = z \end{cases}$$

cylindrical unit vectors

$$\begin{aligned} \mathbf{a}_r &= \frac{\partial \mathbf{r}}{\partial r} = (\cos \theta, \sin \theta, 0) & \mathbf{e}_r &= \frac{\mathbf{a}_r}{|\mathbf{a}_r|} = (\cos \theta, \sin \theta, 0) \\ \mathbf{a}_\theta &= \frac{\partial \mathbf{r}}{\partial \theta} = (-r \sin \theta, r \cos \theta, 0) & \mathbf{e}_\theta &= \frac{\mathbf{a}_\theta}{|\mathbf{a}_\theta|} = (-\sin \theta, \cos \theta, 0) \\ \mathbf{a}_z &= \frac{\partial \mathbf{r}}{\partial z} = (0, 0, 1) & \mathbf{e}_z &= \frac{\mathbf{a}_z}{|\mathbf{a}_z|} = (0, 0, 1) \end{aligned}$$

derivatives of unit vectors

$$\begin{aligned} \frac{\partial \mathbf{e}_r}{\partial \theta} &= (-\sin \theta, \cos \theta, 0) = \mathbf{e}_\theta \\ \frac{\partial \mathbf{e}_\theta}{\partial \theta} &= (-\cos \theta, -\sin \theta, 0) = -\mathbf{e}_r \end{aligned}$$

$$\begin{aligned} \dot{\mathbf{e}}_r &= \frac{\partial \mathbf{e}_r}{\partial \theta} \dot{\theta} = \dot{\theta} \mathbf{e}_\theta \\ \dot{\mathbf{e}}_\theta &= \frac{\partial \mathbf{e}_\theta}{\partial \theta} \dot{\theta} = -\dot{\theta} \mathbf{e}_r \end{aligned}$$

position, velocity and acceleration vectors

$$\begin{aligned}
 \mathbf{r} &= r\mathbf{e}_r + z\mathbf{e}_z \\
 \mathbf{v} &= \dot{\mathbf{r}} = (r\mathbf{e}_r + z\mathbf{e}_z)' = \dot{r}\mathbf{e}_r + r\underbrace{\dot{\mathbf{r}}}_{\dot{\theta}\mathbf{e}_\theta} + \dot{z}\mathbf{e}_z + z\underbrace{\dot{\mathbf{e}}_z}_{=0} \\
 &= \underbrace{\dot{r}}_{v_r}\mathbf{e}_r + \underbrace{r\dot{\theta}}_{v_\theta}\mathbf{e}_\theta + \underbrace{\dot{z}}_{v_z}\mathbf{e}_z \\
 \mathbf{a} &= (\dot{r}\mathbf{e}_r + r\dot{\theta}\mathbf{e}_\theta + \dot{z}\mathbf{e}_z)' \\
 &= (\ddot{r}\mathbf{e}_r + \dot{r}\underbrace{\dot{\mathbf{e}}_r}_{\dot{\theta}\mathbf{e}_\theta} + \dot{r}\dot{\theta}\mathbf{e}_\theta + r\ddot{\theta}\mathbf{e}_\theta + r\dot{\theta}\underbrace{\dot{\mathbf{e}}_\theta}_{-\dot{\theta}\mathbf{e}_r} + \ddot{z}\mathbf{e}_z) \\
 &= \underbrace{(\ddot{r} - r\dot{\theta}^2)}_{a_r}\mathbf{e}_r + \underbrace{(r\ddot{\theta} + 2\dot{r}\dot{\theta})}_{a_\theta}\mathbf{e}_\theta + \underbrace{\ddot{z}}_{a_z}\mathbf{e}_z
 \end{aligned}$$

Frenét coordinates

path - oriented curve with an origin O^* ,

position (natural parameter) of a particle - $\overset{+}{-}$ distance from the particle to origin O^* , measured along the path,

No representation for the position vector.

Velocity vector

$$\mathbf{v} = \frac{d\mathbf{r}}{dt} = \underbrace{\frac{d\mathbf{r}}{ds}}_{\mathbf{e}_t} \frac{ds}{dt} = \underbrace{\dot{s}}_v \mathbf{e}_t$$

\mathbf{e}_t - unit vector, tangent to the path, with orientation consistent with the path

Acceleration vector

$$\mathbf{a} = \frac{d\mathbf{v}}{dt} = \ddot{s}\mathbf{e}_t + \dot{s}\dot{\mathbf{e}}_t = \ddot{s}\mathbf{e}_t + \dot{s}\frac{d\mathbf{e}_t}{ds}\dot{s}$$

$\mathbf{e}_t^2 = 1$ implies $2\mathbf{e}_t \cdot \frac{d\mathbf{e}_t}{ds} = 0$, i.e. $\frac{d\mathbf{e}_t}{ds}$ is perpendicular to the tangent line.

Introducing:

- curvature $\mu = \left| \frac{d\mathbf{e}_t}{ds} \right|$,
- unit (principal) normal vector $\mathbf{e}_n = \frac{\frac{d\mathbf{e}_t}{ds}}{\mu}$,
- radius of curvature $\rho = \mu^{-1}$,

we get,

$$\mathbf{a} = \underbrace{\ddot{s}}_{a_t} \mathbf{e}_t + \underbrace{\frac{\dot{s}^2}{\rho}}_{a_n} \mathbf{e}_n$$

Vectors \mathbf{e}_t , \mathbf{e}_n , and binormal unit vector $\mathbf{e}_b = \mathbf{e}_t \times \mathbf{e}_n$ define Frenét system of coordinates. Velocity has only tangential component, and acceleration has only tangential and principal normal components.

For a planar curve $y = y(x)$,

$$\mu = \frac{|y''|}{[1 + (y')^2]^{\frac{3}{2}}}$$

2 Equations of motion

$$m\mathbf{a} = \mathbf{F}$$

Cartesian coordinates

$$ma_x = m\ddot{x} = F_x$$

$$ma_y = m\ddot{y} = F_y$$

$$ma_z = m\ddot{z} = F_z$$

Cylindrical coordinates

$$ma_r = m(\ddot{r} - r\dot{\theta}^2) = F_r$$

$$ma_\theta = m(r\ddot{\theta} + 2\dot{r}\dot{\theta}) = F_\theta$$

$$ma_z = m\ddot{z} = F_z$$

Frenét coordinates

$$ma_t = m\ddot{s} = F_t$$

$$ma_n = m\frac{\dot{s}^2}{\rho} = F_n$$

$$0 = F_b$$

3 Principle of Work and Energy

Work of a force (field) \mathbf{F} along curve AB

$$U_{12} = \int_{AB} \mathbf{F} \cdot d\mathbf{r} = \int_{u_1}^{u_2} \left(\mathbf{F} \cdot \frac{d\mathbf{R}}{du} \right) du$$

where $\mathbf{r} = \mathbf{r}(u)$, $u_1 \leq u \leq u_2$, $\mathbf{r}(u_1) = A$, $\mathbf{r}(u_2) = B$ is a parameterization for the curve.

Single particle

$$\begin{aligned} a_t &= \frac{dv}{dt}, \quad v = \frac{ds}{dt} \Rightarrow a_t ds = v dv \\ mv dv &= \underbrace{ma_t}_{F_t} ds = F_t ds = \mathbf{F} \cdot d\mathbf{r} \\ \int_{v_1}^{v_2} mv dv &= \int_{\mathbf{r}_1}^{\mathbf{r}_2} \mathbf{F} \cdot d\mathbf{r} \\ \underbrace{\frac{mv_2^2}{2}}_{T_2} - \underbrace{\frac{mv_1^2}{2}}_{T_1} &= \underbrace{\int_{\mathbf{r}_1}^{\mathbf{r}_2} \mathbf{F} \cdot d\mathbf{r}}_{U_{12}} \\ T_1 + U_{12} &= T_2 \end{aligned}$$

System of particles

$$\sum \frac{m_i v_{i,1}^2}{2} + \sum U_{i,12}^{\text{active}} + \sum U_{i,12}^{\text{reactive}} = \sum \frac{m_i v_{i,2}^2}{2}$$

For particles connected with inextensible cables or links, the work of reactive forces can be neglected.

Case of two particles connected with a rigid link

$$\begin{aligned} \mathbf{R}_A &= -R \frac{\mathbf{AB}}{|\mathbf{AB}|}, \quad \mathbf{R}_B = R \frac{\mathbf{AB}}{|\mathbf{AB}|} \\ \mathbf{AB}^2 &= (\mathbf{r}_B - \mathbf{r}_A)^2 = \text{const} \\ 2\mathbf{AB} \cdot (d\mathbf{r}_B - d\mathbf{r}_A) &= 0 \\ \underbrace{R \frac{\mathbf{AB}}{|\mathbf{AB}|}}_{\mathbf{R}_B} \cdot d\mathbf{r}_B + \underbrace{\left(-R \frac{\mathbf{AB}}{|\mathbf{AB}|} \right)}_{\mathbf{R}_A} \cdot d\mathbf{r}_A &= 0 \end{aligned}$$

4 Conservative Forces

A force field \mathbf{F} is conservative if the work done by \mathbf{F} along *any* closed path is zero. If \mathbf{F} is defined in a simply connected domain then the following conditions are equivalent to each

other.

i \mathbf{F} is conservative,

ii There exists a potential (energy) V such that

$$\mathbf{F} = -\nabla V$$

iii $\nabla \times \mathbf{F} = \mathbf{0}$.

Cartesian coordinates

$$\begin{aligned}\nabla V &= \frac{\partial V}{\partial x} \mathbf{i} + \frac{\partial V}{\partial y} \mathbf{j} + \frac{\partial V}{\partial z} \mathbf{k} \\ \nabla \times \mathbf{F} &= \left(\frac{\partial F_3}{\partial x_2} - \frac{\partial F_2}{\partial x_3}, \frac{\partial F_1}{\partial x_3} - \frac{\partial F_3}{\partial x_1}, \frac{\partial F_2}{\partial x_1} - \frac{\partial F_1}{\partial x_2} \right)\end{aligned}$$

Polar coordinates

$$\nabla V = \frac{\partial V}{\partial r} \mathbf{e}_r + \frac{1}{r} \frac{\partial V}{\partial \theta} \mathbf{e}_\theta$$

Examples of conservative forces

Weight

$$\mathbf{F} = (0, 0, -W)$$

$$V = Wz + c$$

Gravitational force

$$\begin{aligned}\mathbf{F} &= -\frac{GMm}{r^2} \frac{\mathbf{r}}{r} \\ V &= \frac{GMm}{r} + c\end{aligned}$$

Spring force

$$\begin{aligned}\mathbf{F} &= -k\Delta l \frac{\mathbf{r}}{r} \\ V &= \frac{k\Delta l^2}{2} + c\end{aligned}$$

Here $\mathbf{r} = (x, y, z)$ is position vector, $r = |\mathbf{r}|$, and $\Delta = r - l_0$ is the stretch of the spring (l_0 denotes the unstretched length).

5 Principle of Linear Impulse and Momentum

Single particle

$$m\dot{\mathbf{v}} = \mathbf{F}$$

$$m\mathbf{v}_2 - m\mathbf{v}_1 = \int_{t_1}^{t_2} \mathbf{F} dt$$

$$\underbrace{m\mathbf{v}_1}_{\text{initial momentum}} + \underbrace{\int_{t_1}^{t_2} \mathbf{F} dt}_{\text{impulse}} = \underbrace{m\mathbf{v}_2}_{\text{final momentum}}$$

Frenét coordinates version

$$m\dot{v} = F$$

$$mv_2 - mv_1 = \int_{t_1}^{t_2} F_t dt$$

$$\underbrace{mv_1}_{\text{initial momentum}} + \underbrace{\int_{t_1}^{t_2} F_t dt}_{\text{tangential impulse}} = \underbrace{mv_2}_{\text{final momentum}}$$

System of particles

$$\underbrace{\sum m_i \mathbf{v}_{i,1}}_{\text{initial momentum}} + \underbrace{\int_{t_1}^{t_2} \mathbf{F}_i^{\text{active}} dt}_{\text{impulse}} = \underbrace{\sum m_i \mathbf{v}_{i,2}}_{\text{final momentum}}$$

6 Principle of Angular Impulse and Momentum

Single particle *A*. Point *O* fixed, $\mathbf{r} = \mathbf{OA}$

$$m\dot{\mathbf{v}} = \mathbf{F}$$

$$\mathbf{r} \times m\dot{\mathbf{v}} = \mathbf{r} \times \mathbf{F}$$

But

$$(\mathbf{r} \times m\mathbf{v})' = \underbrace{\dot{\mathbf{r}}}_{\mathbf{v}} \times m\mathbf{v} + \mathbf{r} \times m\dot{\mathbf{v}} = \mathbf{r} \times m\dot{\mathbf{v}},$$

so

$$\underbrace{(\mathbf{r} \times m\mathbf{v})'}_{\text{angular momentum } \mathbf{H}_O} = \underbrace{\mathbf{r} \times \mathbf{F}}_{\text{moment } \mathbf{M}_O}$$

$$\underbrace{\mathbf{r} \times m\mathbf{v}_1}_{\text{initial angular momentum}} + \underbrace{\int_{t_1}^{t_2} \mathbf{M}_O(t) dt}_{\text{angular impulse}} = \underbrace{\mathbf{r} \times m\mathbf{v}_2}_{\text{final angular momentum}}$$

System of particles. Point O fixed, $\mathbf{r}_i = \mathbf{O}\mathbf{A}_i$

$$\underbrace{\left(\sum_i \mathbf{r}_i \times m_i \mathbf{v}_i \right)'}_{\mathbf{H}_O} = \underbrace{\sum_i \mathbf{r}_i \times \mathbf{F}_i}_{\mathbf{M}_O}$$

$$\mathbf{H}_O^{\text{initial}} + \int_{t_1}^{t_2} \mathbf{M}_O(t) dt = \mathbf{H}_O^{\text{final}}$$

System of particles. Point O - fixed, C - center of mass

$$\begin{aligned} \mathbf{H}_C &= \sum_i \mathbf{C}\mathbf{A}_i \times m_i \mathbf{v}_i = \sum_i (\mathbf{C}\mathbf{O} + \mathbf{O}\mathbf{A}_i) \times m_i \mathbf{v}_i \\ &= \mathbf{C}\mathbf{O} \times \sum_i m_i \mathbf{v}_i + \sum_i \mathbf{O}\mathbf{A}_i \times m_i \mathbf{v}_i \end{aligned}$$

From the definition of center of mass, $M = \sum_i m_i$ - total mass,

$$\begin{aligned} \mathbf{O}\mathbf{C} &= \frac{\sum_i m_i \mathbf{O}\mathbf{A}_i}{M} \\ M \mathbf{O}\mathbf{C} &= \sum_i m_i \mathbf{O}\mathbf{A}_i \quad / \frac{d}{dt} \\ M \mathbf{v}_c &= \sum_i m_i \mathbf{v}_i \end{aligned}$$

So

$$\begin{aligned} \mathbf{H}_C &= \mathbf{C}\mathbf{O} \times M \mathbf{v}_c + \mathbf{H}_O \quad / \frac{d}{dt} \\ \dot{\mathbf{H}}_C &= -\mathbf{v}_c \times M \mathbf{v}_c + \mathbf{C}\mathbf{O} \times M \dot{\mathbf{v}}_c + \dot{\mathbf{H}}_O \\ &= \mathbf{C}\mathbf{O} \times \sum_i \mathbf{F}_i + \sum_i \mathbf{O}\mathbf{A}_i \times \mathbf{F}_i \\ &= \sum_i (\mathbf{C}\mathbf{O} + \mathbf{O}\mathbf{A}_i) \times \mathbf{F}_i \\ &= \sum_i \mathbf{C}\mathbf{A}_i \times \mathbf{F}_i = \mathbf{M}_C \end{aligned}$$

7 Kinematics of a Rigid Body

For arbitrary points P and A on the rigid body

$$\mathbf{v}_P = \mathbf{v}_A + \underbrace{\boldsymbol{\omega}}_{\text{angular velocity vector}} \times \mathbf{AP}$$

$$\mathbf{a}_P = \mathbf{a}_A + \underbrace{\boldsymbol{\alpha}}_{\text{angular acceleration vector}} \times \mathbf{AP} + \boldsymbol{\omega} \times (\boldsymbol{\omega} \times \mathbf{AP})$$

8 Moving Frame of Reference

The frame of reference is identified with a Cartesian system with origin at point A , angular velocity vector $\boldsymbol{\Omega}$, and unit vectors \mathbf{e}_i .

Time derivative of a vector-valued function $\mathbf{f}(t) = \sum_i f_i(t) \mathbf{e}_i(t)$.

$$\begin{aligned} \dot{\mathbf{f}} &= \sum_i \dot{f}_i \mathbf{e}_i + \sum_i f_i \underbrace{\dot{\mathbf{e}}_i}_{\boldsymbol{\Omega} \times \mathbf{e}_i} \\ &= \sum_i \dot{f}_i \mathbf{e}_i + \boldsymbol{\Omega} \times \left(\sum_i f_i \mathbf{e}_i \right) \\ &= \underbrace{\dot{\mathbf{f}}|_{\text{rel}}}_{\text{relative time derivative}} + \boldsymbol{\Omega} \times \mathbf{f} \end{aligned}$$

Velocity and acceleration

$$\begin{aligned} \mathbf{OP} &= \mathbf{OA} + \mathbf{AP} \\ \mathbf{v}_P &= \mathbf{v}_A + \underbrace{\dot{\mathbf{AP}}|_{\text{rel}}}_{\mathbf{v}_{P,\text{rel}}} + \boldsymbol{\Omega} \times \mathbf{AP} \\ \mathbf{a}_P &= \mathbf{a}_A + \underbrace{\dot{\mathbf{v}}_{P,\text{rel}}|_{\text{rel}}}_{\mathbf{a}_{P,\text{rel}}} + \boldsymbol{\Omega} \times \mathbf{v}_{P,\text{rel}} + \dot{\boldsymbol{\Omega}} \times \mathbf{AP} + \boldsymbol{\Omega} \times (\mathbf{v}_{P,\text{rel}} + \boldsymbol{\Omega} \times \mathbf{AP}) \\ &= \underbrace{\mathbf{a}_A + \dot{\boldsymbol{\Omega}} \times \mathbf{AP} + \boldsymbol{\Omega} \times (\boldsymbol{\Omega} \times \mathbf{AP})}_{\text{rigid body acceleration}} + \underbrace{\mathbf{a}_{P,\text{rel}}}_{\text{relative acceleration}} + \underbrace{2\boldsymbol{\Omega} \times \mathbf{v}_{P,\text{rel}}}_{\text{Coriolis acceleration}} \end{aligned}$$

9 Rigid Body Kinetics

Tensor(matrix) of inertia. For O fixed, or $O = C$, center of mass,

$$\mathbf{H}_O = \underbrace{\mathbf{I}_O}_{\text{inertia tensor}} \boldsymbol{\omega}$$

Equation of rotational motion for a rigid body. For O fixed, or $O = C$, center of mass,

$$\dot{\mathbf{H}}_O = \mathbf{M}_O \quad \text{implies} \quad (\mathbf{I}_O \boldsymbol{\omega})' = \mathbf{M}_O$$

In a system of coordinates rotating with angular velocity vector $\boldsymbol{\Omega}$, in which $\dot{\mathbf{I}}_O = \mathbf{0}$,

$$\mathbf{I}_O \dot{\boldsymbol{\omega}}|_{\text{rel}} + \boldsymbol{\Omega} \times (\mathbf{I}_O \boldsymbol{\omega}) = \mathbf{M}_O$$

In 2D, $\mathbf{I}_O \boldsymbol{\omega} = (0, 0, I_O \omega)$, $\mathbf{M}_O = (0, 0, M_O)$ imply

$$I_O \alpha = M_O$$

10 Kinetic Energy of a Rigid Body

Arbitrary motion

$$\begin{aligned} T &= \frac{1}{2} \sum_i m_i (\mathbf{v}_C + \boldsymbol{\omega} \times \mathbf{C}P_i)^2 \\ &= \frac{1}{2} \sum_i m_i (\mathbf{v}_C^2 + 2\mathbf{v}_C \circ (\boldsymbol{\omega} \times \mathbf{C}P_i) + (\boldsymbol{\omega} \times \mathbf{C}P_i)^2) \\ &= \frac{1}{2} \sum_i m_i v_C^2 + \mathbf{v}_C \circ \boldsymbol{\omega} \times \underbrace{\sum_i m_i \mathbf{C}P_i}_0 + \frac{1}{2} \sum_i m_i (\boldsymbol{\omega} \times \mathbf{C}P_i)^2 \end{aligned}$$

In 2D,

$$(\boldsymbol{\omega} \times \mathbf{C}P_i)^2 = (x_i^2 + y_i^2) \omega^2$$

so,

$$\sum_i m_i (\boldsymbol{\omega} \times \mathbf{C}P_i)^2 = \underbrace{\sum_i m_i (x_i^2 + y_i^2)}_{I_C} \omega^2$$

and,

$$T = \frac{1}{2} M v_c^2 + \frac{1}{2} I_C \omega^2$$

Rotation about a fixed point O , $d = |\mathbf{OC}|$.

$$\begin{aligned} T &= \frac{1}{2} M v_C^2 + \frac{1}{2} I_C \omega^2 \\ &= \frac{1}{2} M (\omega d)^2 + \frac{1}{2} I_C \omega^2 \\ &= \frac{1}{2} \underbrace{(M d^2 + I_C)}_{I_O} \omega^2 \end{aligned}$$